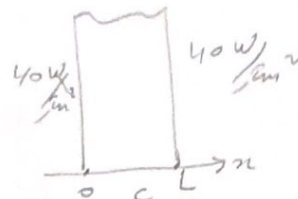
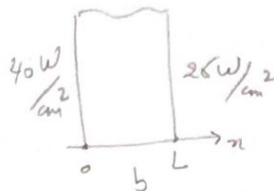
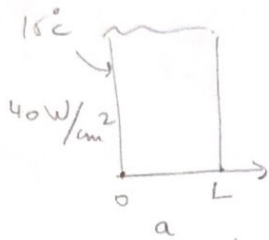


Consigne type Interrogation écrite Groupe 1

1/ types des conditions aux limites pour chaque cas



a) pour $x=0$ densité de flux imposée (Neuman) $\Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=0} = \varphi_0 = 40 \text{ W/cm}^2$

pour $x=0$ température imposée (Dirichlet) $\Rightarrow T(0) = T = 15^\circ\text{C}$

b) pour $x=0$ densité de flux imposée (Neuman) $\Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=0} = \varphi_0 = 40 \text{ W/cm}^2$

pour $x=L$ " " " (Neuman) $\Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=L} = \varphi_0 = 28 \text{ W/cm}^2$

c) pour $x=0$ densité de flux imposée (Neuman) $\Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=0} = 40 \text{ W/cm}^2$

pour $x=L$ " " " (Neuman) $\Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=L} = 40 \text{ W/cm}^2$

2/ Détermination de l'expression de la variation de Température (a).

On a pour ce cas. (régime permanent, sans source interne, unidimensionnel)

$$\frac{d^2 T(x)}{dx^2} = 0 \quad \text{--- (1)}$$

$$\text{pour } x=0 \Rightarrow -\lambda \frac{dT(x)}{dx} \Big|_{x=0} = \varphi_0$$

$$x=0 \Rightarrow T(0) = T = 15^\circ\text{C}$$

$$\text{(1)} \Rightarrow \frac{dT(x)}{dx} = A \Rightarrow T(x) = Ax + B$$

$$x=0 \Rightarrow -\lambda \frac{dT(x)}{dx} = -\lambda \cdot A = \varphi_0 \Rightarrow A = -\frac{\varphi_0}{\lambda}$$

$$x=0 \quad T(0) = T = 15^\circ\text{C} \Rightarrow \boxed{T(x) = -\frac{\varphi_0}{\lambda} x + 15}$$

Consigne type interrogation groupe 1

a) le taux de transfert de chaleur

$$\frac{\phi}{S} = \varphi = \frac{\Delta T}{R_{eq}} \quad (1)$$

$$R_{eq} = R_{eq}^i + R_{cond_1} + R_{cond_4} \quad (1)$$

$$\frac{1}{R_{eq}^i} = \frac{1}{R_{cond_2}} + \frac{1}{R_{cond_3}} \quad (1)$$

$$R_{cond_2} = \frac{e_2}{\lambda_2} = \frac{7,5 \cdot 10^{-2}}{0,76} = 9,86 \cdot 10^{-2} \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,5)$$

$$R_{cond_3} = \frac{e_3}{\lambda_3} = \frac{7,5 \cdot 10^{-2}}{0,69} = 10,86 \cdot 10^{-2} \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,6)$$

$$\frac{1}{R_{eq}^i} = 20,72 \Rightarrow R_{eq}^i = 0,0482 \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,1)$$

$$R_{cond_1} = \frac{e_1}{\lambda_1} = \frac{2,5 \cdot 10^{-2}}{0,11} = 22,72 \cdot 10^{-2} \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,6)$$

$$R_{cond_4} = \frac{e_4}{\lambda_4} = \frac{5 \cdot 10^{-2}}{0,147} = 34,01 \cdot 10^{-2} \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,6)$$

$$R_{eq} = 0,0482 + 34,01 \cdot 10^{-2} + 22,72 \cdot 10^{-2} = 0,6158 \text{ } ^\circ\text{C m}^2/\text{W} \quad (0,5)$$

$$\varphi = \frac{\Delta T}{R_{eq}} = \frac{(370 - 66)}{0,6158} = 493,90 \text{ W/m}^2 \quad (0,6)$$

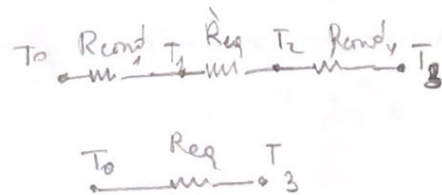
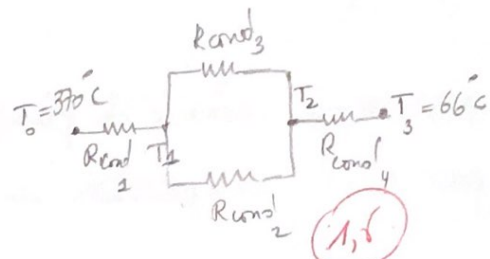
b) calcul des Température T_1 et T_2

$$\varphi = \frac{\Delta T_1}{R_{cond_1}} \Rightarrow \Delta T_1 = \varphi \cdot R_{cond_1} \Rightarrow T_1 = 370 - 493,9 \cdot 22,72 \cdot 10^{-2} = 257,78^\circ\text{C} \quad (0,1)$$

$$T_1 = 257,78^\circ\text{C} \quad (0,1)$$

$$\varphi = \frac{\Delta T_2}{R_{cond_2}} \Rightarrow \Delta T_2 = \varphi \cdot R_{cond_2} \Rightarrow T_2 = 66 + \varphi \cdot R_{cond_2} = 66 + 493,9 \cdot 9,86 \cdot 10^{-2} \quad (0,1)$$

$$T_2 = 233,97^\circ\text{C} \quad (0,1)$$



congé type interrogation groupe 3'

$$\lambda_{\text{verre}} = 0,78 \text{ W/m}\cdot\text{C}$$

$$\lambda_{\text{air}} = 0,026 \text{ W/m}\cdot\text{C}$$

$$h_1 = 10 \text{ W/m}^2\cdot\text{C}, h_2 = 25 \text{ W/m}^2\cdot\text{C}$$

$$e_{\text{air}} = 12 \cdot 10^{-3} \text{ m}, e_{\text{verre}} = 3 \cdot 10^{-3} \text{ m}$$

$$S = 1,2 \times 2 = 2,4 \text{ m}^2$$

1/ Détermination du taux de transfert de chaleur

$$\phi = \frac{\Delta T}{R_{\text{eq}}} \quad (1,5)$$

$$R_{\text{eq}} = R_{\text{conv}_1} + R_{\text{cond}_1} + R_{\text{cond}_2} + R_{\text{cond}_3} + R_{\text{conv}_2} \quad (1,5)$$

$$R_{\text{conv}_1} = \frac{1}{h_1 \cdot S} = \frac{1}{10 \times (1,2 \times 2)} = 0,041 \text{ } ^\circ\text{C/W} \quad (0,5)$$

$$R_{\text{conv}_2} = \frac{1}{h_2 \cdot S} = \frac{1}{25 \times (1,2 \times 2)} = 0,0167 \text{ } ^\circ\text{C/W} \quad (0,5)$$

$$R_{\text{cond}_1} = \frac{e_1}{\lambda_1 \cdot S} = \frac{3 \cdot 10^{-3}}{0,78 \times (1,2 \times 2)} = 0,0016 \text{ } ^\circ\text{C/W} \quad (0,5)$$

$$R_{\text{cond}_2} = \frac{e_2}{\lambda_2 \cdot S} = \frac{12 \cdot 10^{-3}}{0,026 \times (2 \times 1,2)} = 0,192 \text{ } ^\circ\text{C/W} \quad (0,5)$$

$$R_{\text{cond}_3} = \frac{e_3}{\lambda_3 \cdot S} = \frac{e_1}{\lambda_1 \cdot S} = R_{\text{cond}_1} = 0,0016 \text{ } ^\circ\text{C/W} \quad (0,5)$$

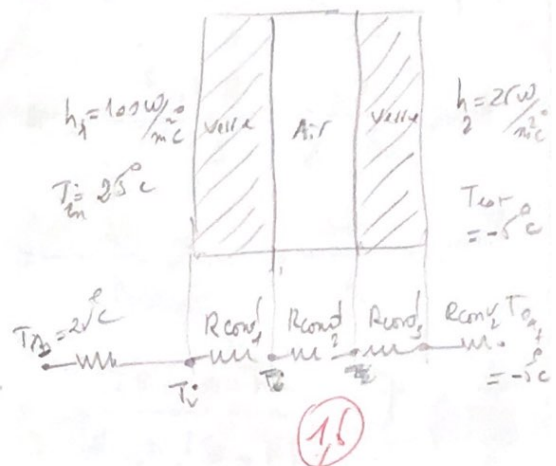
$$R_{\text{eq}} = 0,2529 \quad (0,5)$$

$$\phi = \frac{\Delta T}{R_{\text{eq}}} = \frac{24 - (-5)}{0,2529} = 114,67 \text{ W} \quad (0,5)$$

2/ Calcul de la Température de la surface interne. (T_i)

$$\phi = \frac{\Delta T}{R_{\text{conv}_1}} = \frac{T_1 - T_i}{R_{\text{conv}_1}} \Rightarrow T_i = T_1 - \phi \cdot R_{\text{conv}_1} = 24 - 114,67 \cdot 0,041 \quad (0,5)$$

$$T_i = 19,30 \text{ } ^\circ\text{C} \quad (0,5)$$



corrigé type interrogation écrite groupe 2

• calcul du flux (densité de flux).

$$\varphi = \frac{\Phi}{S} = \frac{\Delta T}{R_{eq}} \quad (0,5)$$

$$R_{eq} = R_{cond_1} + R_{cond_2} \quad (0,5)$$

$$R_{cond_1} = \frac{e_1}{\lambda_1} = \frac{2 \cdot 10^{-2}}{2} = 1 \cdot 10^{-2} \frac{m^2}{W} \quad (0,5)$$

$$R_{cond_2} = \frac{e_2}{\lambda_2} = \frac{6 \cdot 10^{-2}}{1} = 6 \cdot 10^{-2} \frac{m^2}{W} \quad (0,5)$$

$$\varphi = \frac{\Delta T}{R_{eq}} = \frac{100 - 25}{(1+6) \cdot 10^{-2}} = 1071,43 \frac{W}{m^2} \quad (0,5)$$

calcul des Températures T_i, T_j, T_r

$$T_i = ?$$

$$\varphi = \frac{\Delta T_1}{R_{cond_1}} = \frac{T_1 - T_i}{R_{cond_1}} \Rightarrow \varphi \cdot R_{cond_1} = T_1 - T_i \Rightarrow T_i = T_1 - \varphi \cdot R_{cond_1} = T_1 - \varphi \cdot \frac{e_1}{\lambda_1} \quad (0,5)$$

$$T_i = 100 - 1071,43 \cdot 1 \cdot 10^{-2} = 100 - 1071,43 \cdot 10^{-2} = 89,28^\circ C \quad (0,5)$$

$$T_j = ?$$

$$\varphi = \frac{\Delta T_3}{R_{cond_3}} \quad R_{cond_3} = \frac{e_3}{\lambda_3} = \frac{4 \cdot 10^{-2}}{5} = 0,8 \cdot 10^{-2} \frac{m^2}{W} \quad (0,5)$$

$$T_j = T_2 - \varphi \cdot R_{cond_3} = 25 - 1071,43 \cdot 0,8 \cdot 10^{-2} = 16,42^\circ C \quad (0,5)$$

$$T_r = ?$$

$$\varphi = \frac{\Delta T_4}{R_{cond_4}} \Rightarrow T_j - T_r = \varphi \cdot R_{cond_4} \Rightarrow T_r = T_j - \varphi \cdot R_{cond_4} = 16,42 - 1071,43 \cdot 10^{-2} \quad (0,5)$$

$$R_{cond_4} = \frac{4 \cdot 10^{-2}}{4} = 10^{-2} \quad (0,5)$$

$$T_r = 5,70^\circ C \quad (0,5)$$

